

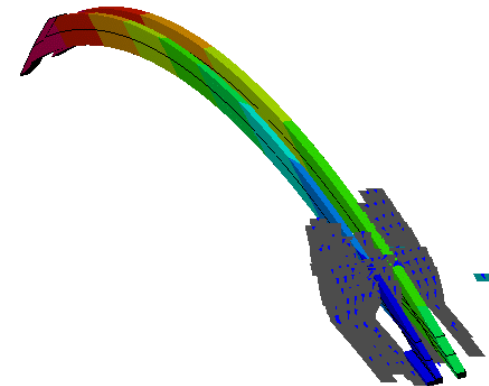
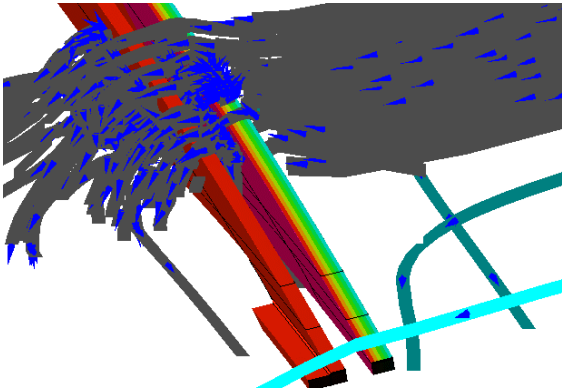


Numerical Simulation of Flow, Pressure and Motion of Front Back Fingers in a Two Rows Finger Seal

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ABSTRACT



- **This proposal fits within the programmatic long-term development direction for turbine engine seals of the Seal Team of the Mechanical Component Branch.**
- **The intended work concerns the further development of the Finger Seal concept which is a compliant passive-adaptive seal meant to mitigate (and eventually replace) the shortcomings of the entire class of rigid seals used today (labyrinth, honeycomb, mechanical face seals) in the gas turbines and compressors.**



GOALS



➤ First,

we are aiming at developing a fully integrated numerical 3-D model, which couples the hydrodynamic fluid model (Navier-Stokes based) to the solid mechanics code that models the compliance of the fingers.

The coupled codes that feedback in an iterative mode, will allow the full simulation of the passive-adaptive properties of this innovative seal.

➤ Secondly,

experimentally, we shall test alternative models of finger seals in an effort to better understand their sealing and lifting properties, as well as guide and validate the code numerical development.

In Year II, in collaboration with the Seal Team of the Mechanical Components Branch, we shall extend the University of Akron based experimental program to the High Temperature Test Rig at NASA Glenn Research Center. This will allow moving our technology readiness level from a room temperature laboratory environment (TRL-4) to the high temperature, engine relevant environment (TRL-5).



NUMERICAL SIMULATION COMPONENT MODULES



⇒ **Mechanical model of the single finger and assembly of fingers.**

This model will entail the generation of a finite element based code that will simulate the stiffness and damping of the element as it is subject to engine environment pressures (high and low side), hydrodynamic pressures at the finger foot/shaft interface, and Coulomb friction between the two rows of fingers.

⇒ **Hydrodynamic fluid model.** This model has to simulate the hydrodynamic lifting effects on the finger seal, as well as the primary and secondary leakages as they occur between the fingers and at the shaft/finger foot interface. We intend to use an already existing numerical package and tailor it to the particular needs of the project.



NUMERICAL SIMULATION COMPONENT MODULES



⇒) **Solid/fluid Interaction with the Dynamics module.** Through the implementation of a) and b) we shall obtain a fully interactive model that will model the interaction between finger mechanics and the 3-D fluid hydrodynamic behavior. In this context we shall generate a complete pressure map of the hydrodynamic pressures ensuing under the finger pad footprint. All external body forces acting on the finger will be accounted for, in this model.

⇒) **Simplified spreadsheet design.** With a), b) and c) implemented we project the possibility that a detailed parametric run will allow creation of a database that can be used for the creation of a simplified calculation methodology that will use a spreadsheet format, without any further need of 3-D calculations.

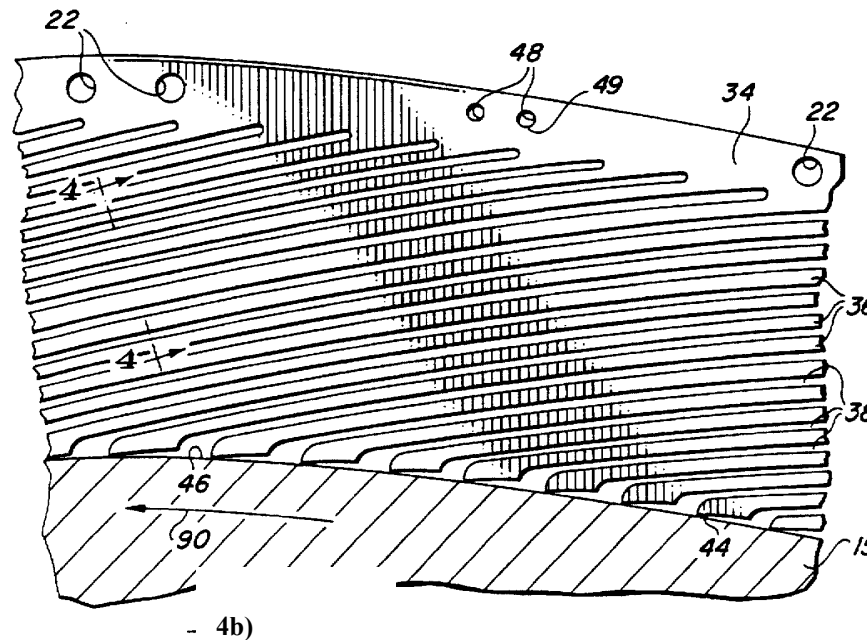
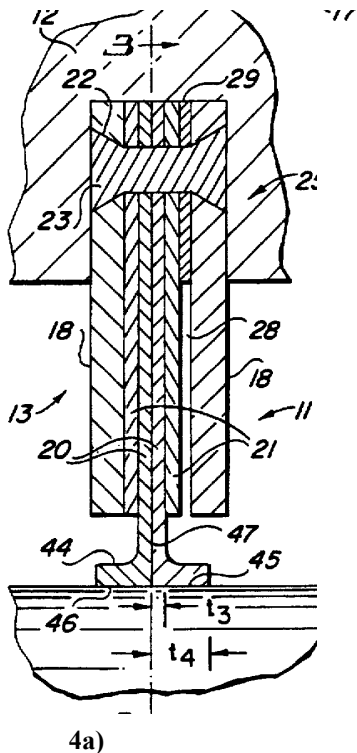


EXPERIMENTAL PROGRAM



⇒ The Tribology Laboratory at the University of Akron possesses a high-speed rig that can be run up to 15,000 rpm. The rig contains all necessary controls and data acquisition system for measuring pressures, temperatures, rotor orbits. The spindle is mounted in cantilever and allows installation of a slip ring at its axial end.

- full pressure and temperature maps
- identification of lift-off and torque characteristics
- high speed visualization of the finger motion and subsequent leakage patterns
- identification of the physics of finger lift-off
- flow visualization of flow patterns before and after finger seal pad lift-off
- effects on sealing efficiency and seal hydrodynamics when
- spiral grooves are etched in the shaft
- grooves are etched on the seal footpads.
- effect of eccentric rotor on seal performance



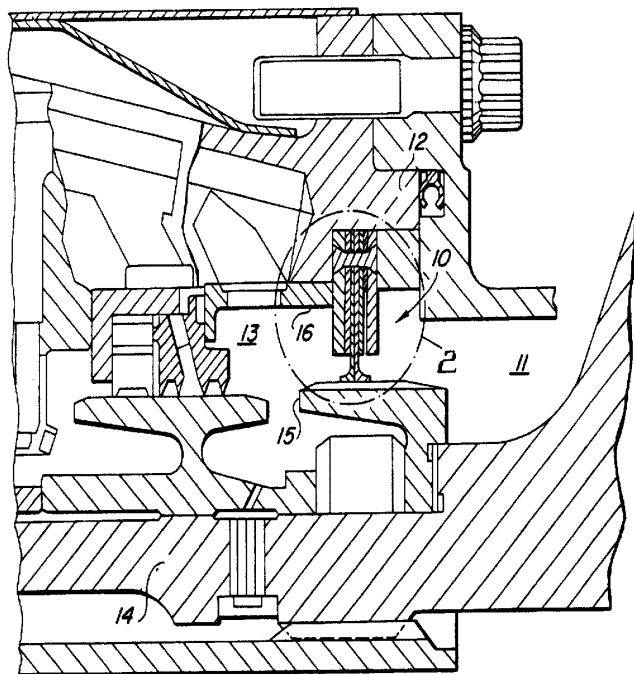
Seal Two row Configuration with Wide Finger Pads. Cross Section and Side View of the Seal
(U.S. Patent No. 5,755,445)

U.S. Patent

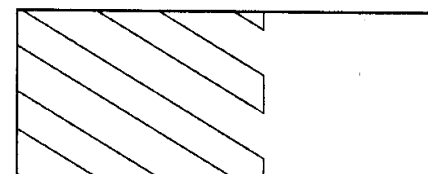
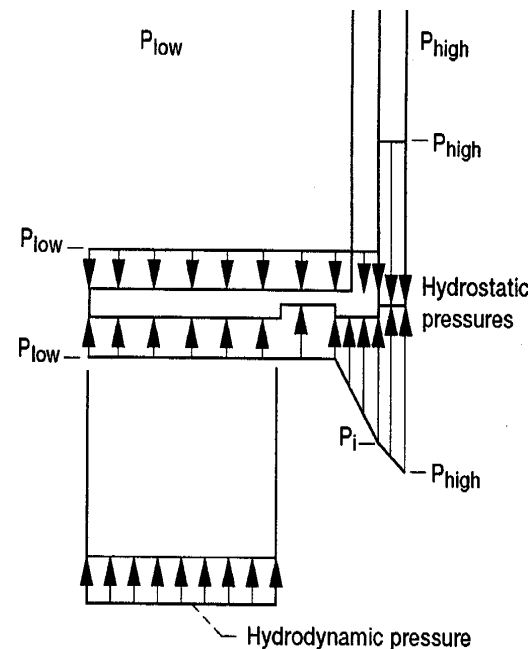
May 26, 1998

Sheet 1 of 3

5,755,445



Typical Application of the Finger Seal presented on previous slide (U.S. Patent No. 5,755,445)



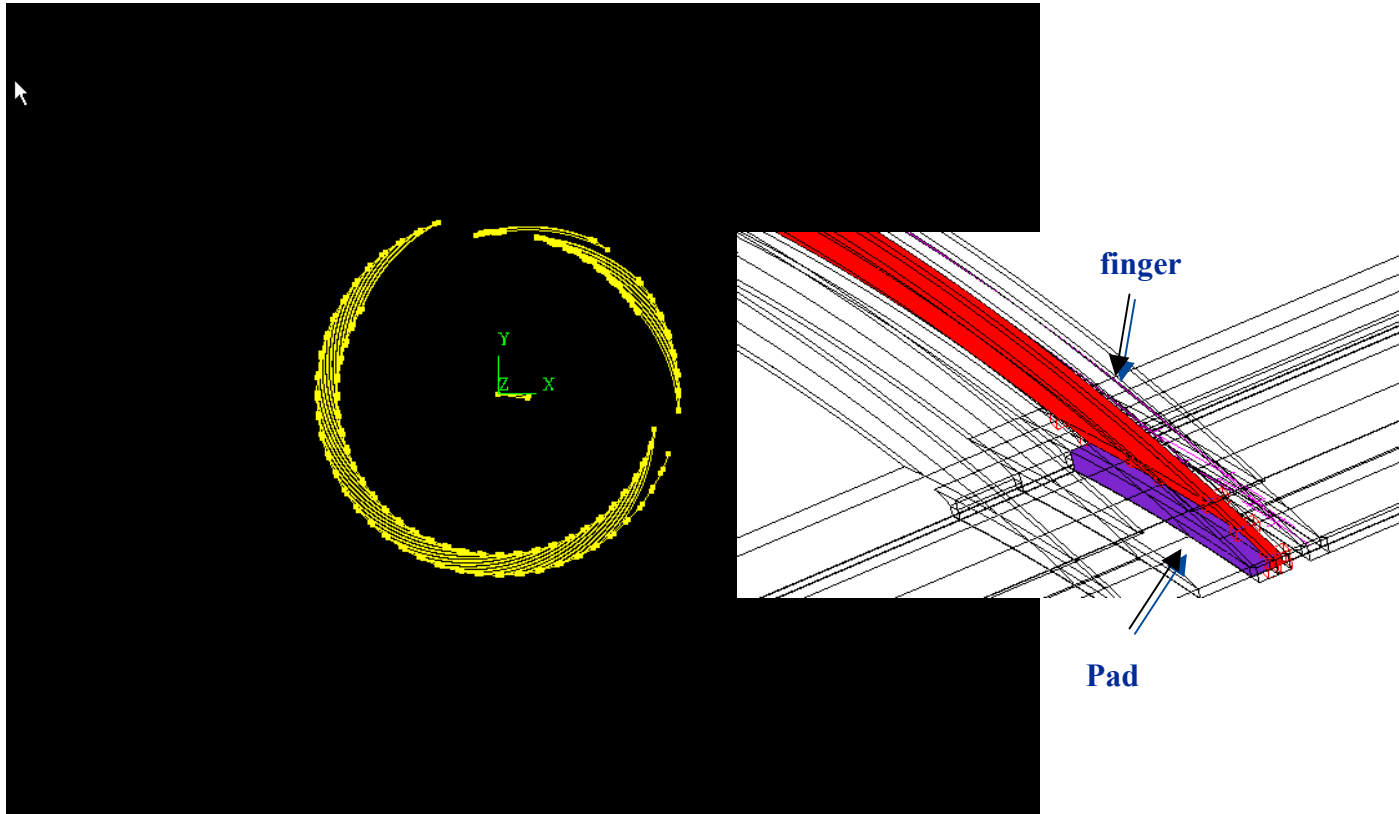
Single Finger as a Free Body Diagram and Geometrical Changes Proposed For Better Wear Behavior



Compliant Fingers Discretization

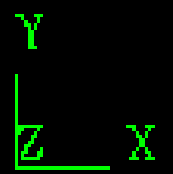
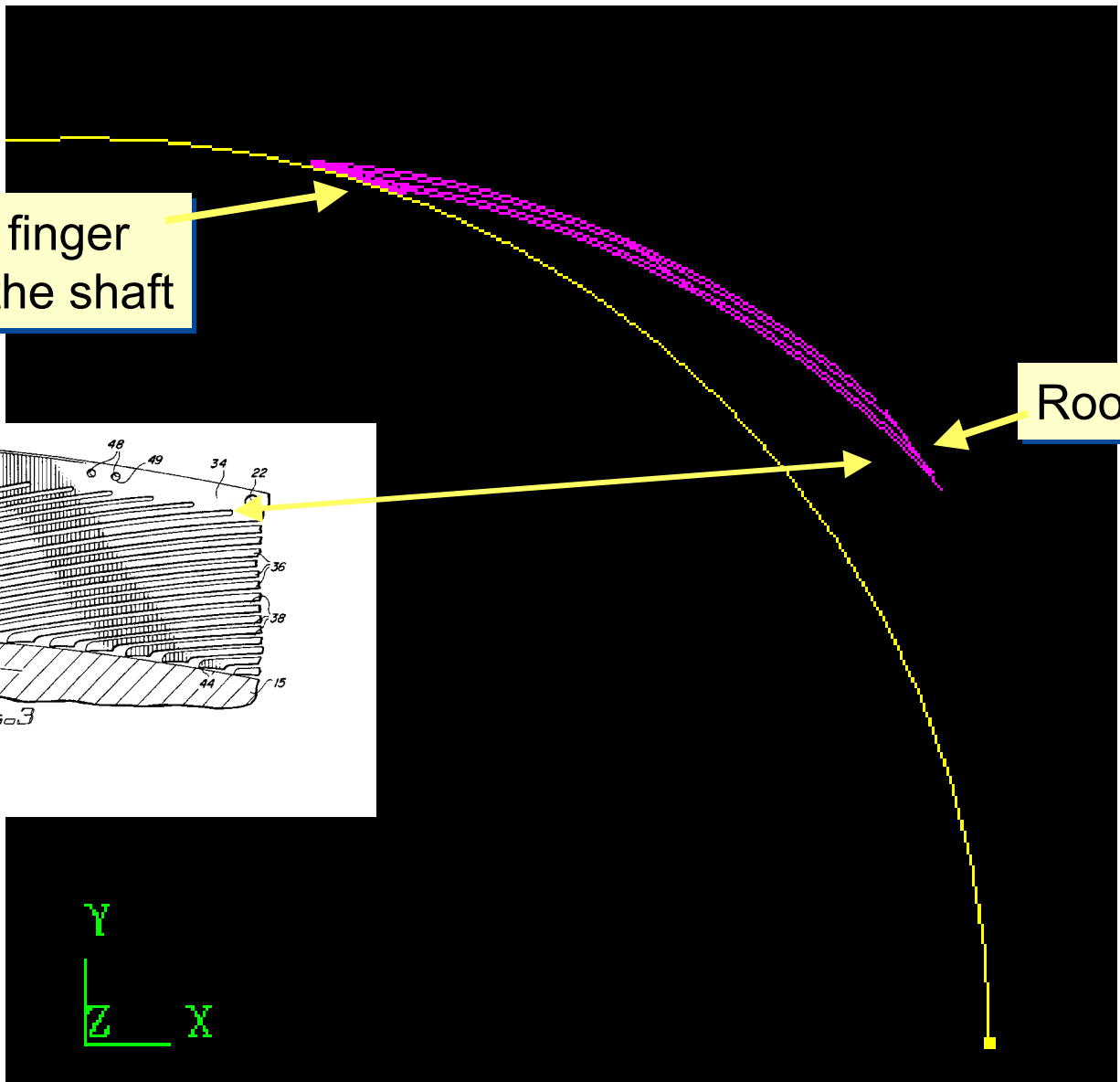
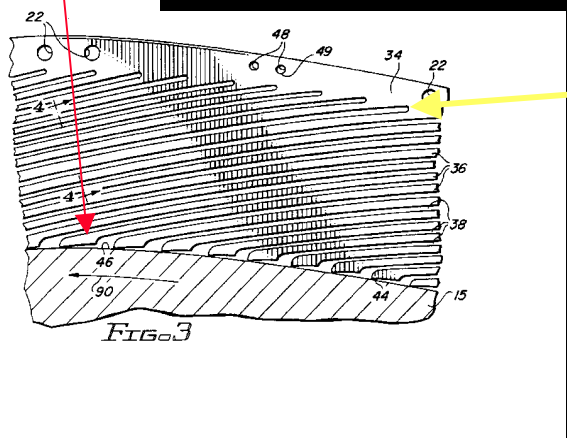


Assembly of 72
fingers along the
circumference



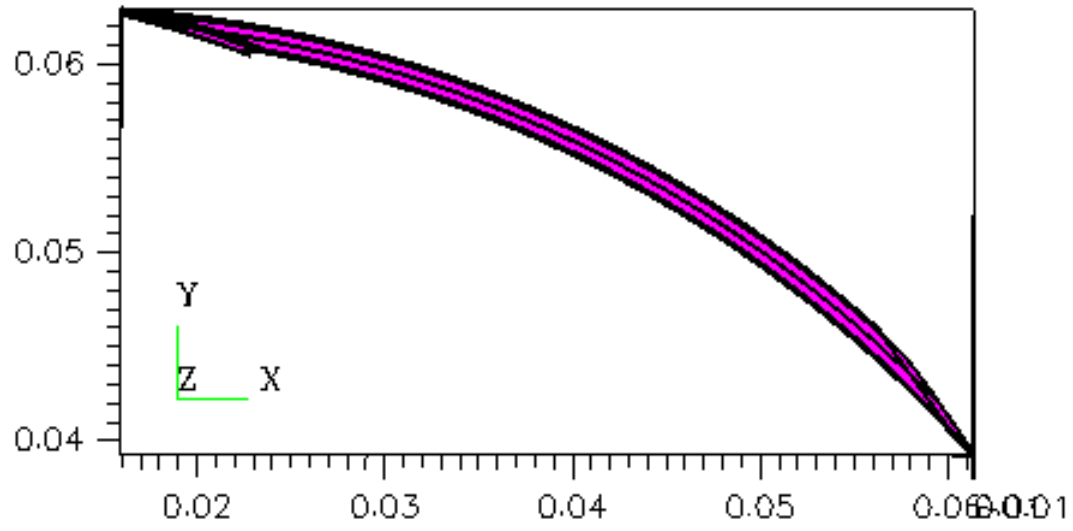
Tip of the finger resting on the shaft

Root of the Finger



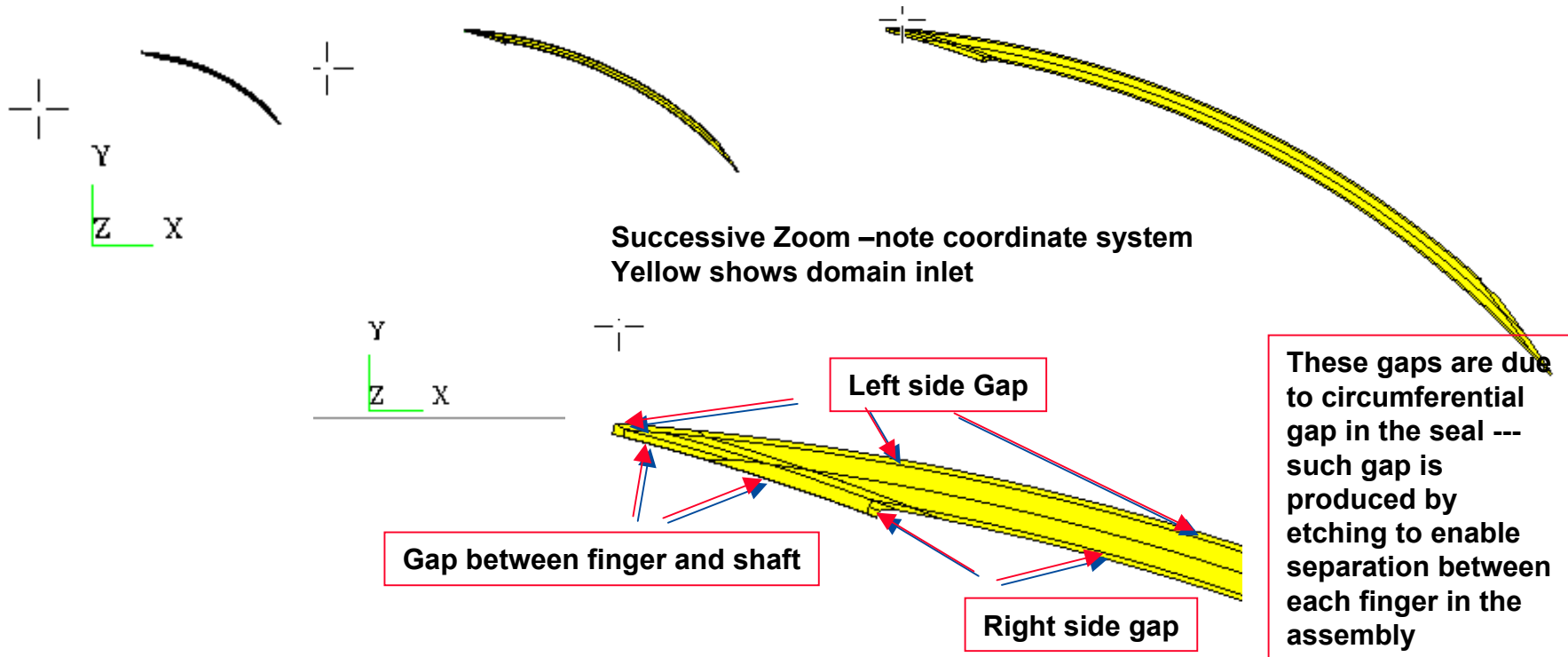


Front View– Geometry, single finger





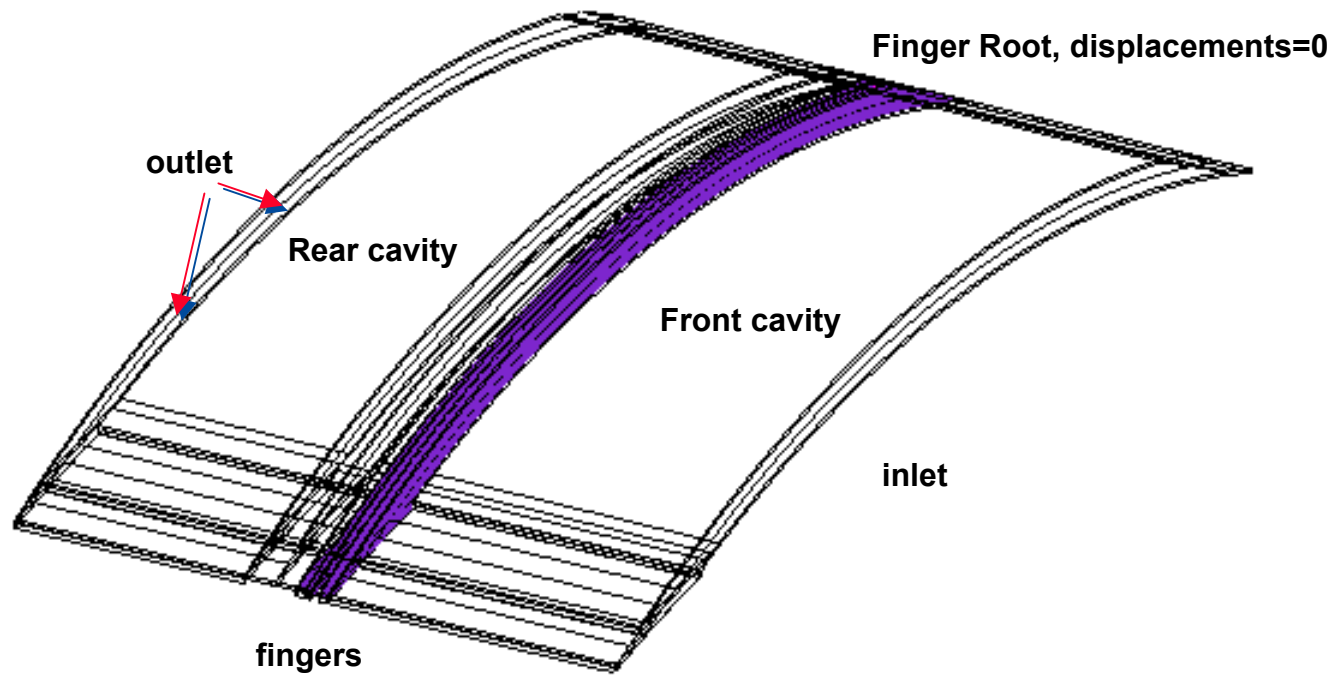
Front View Sequence –One Finger



This slide shows one finger out of 72 located along the circumference. Z is axis along the shaft.

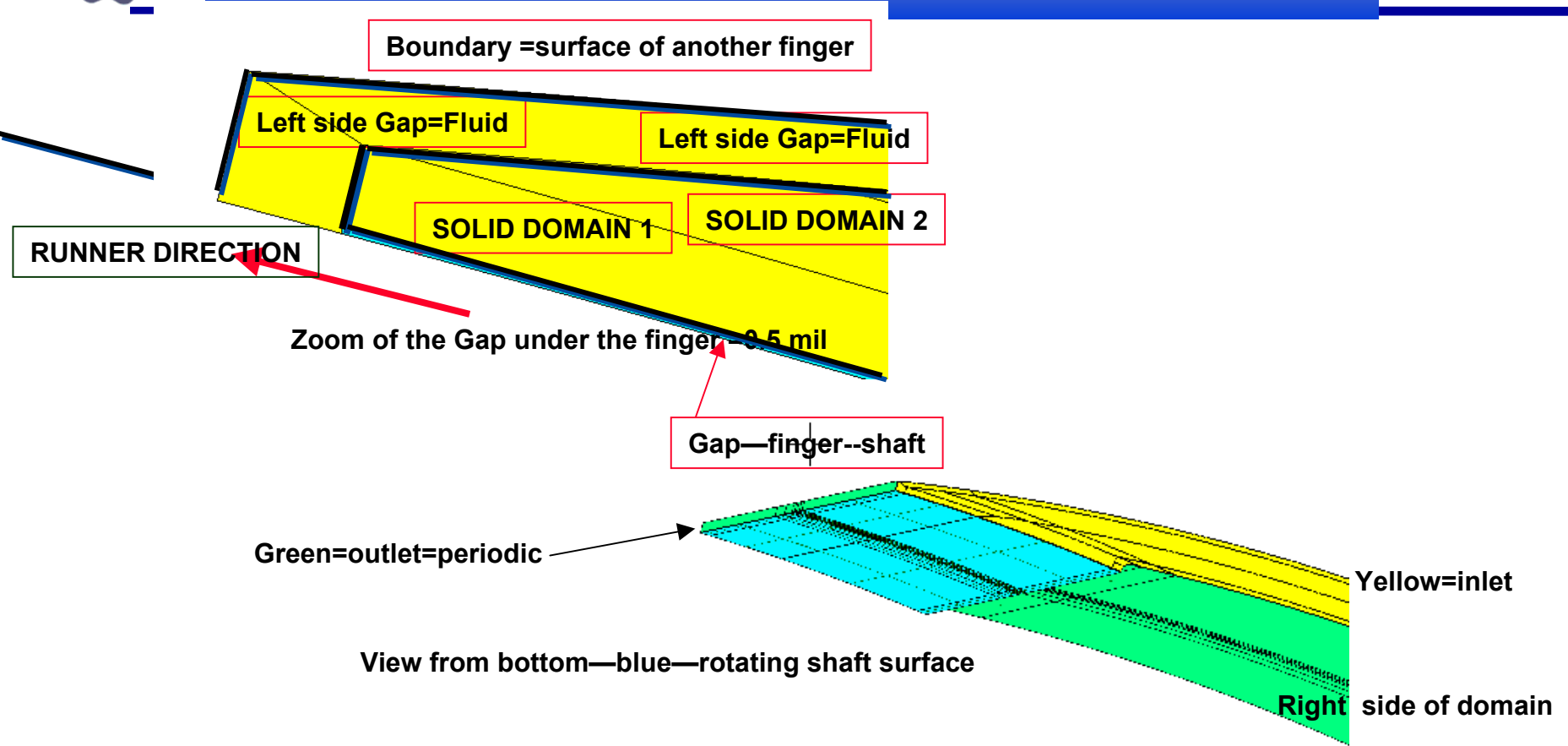


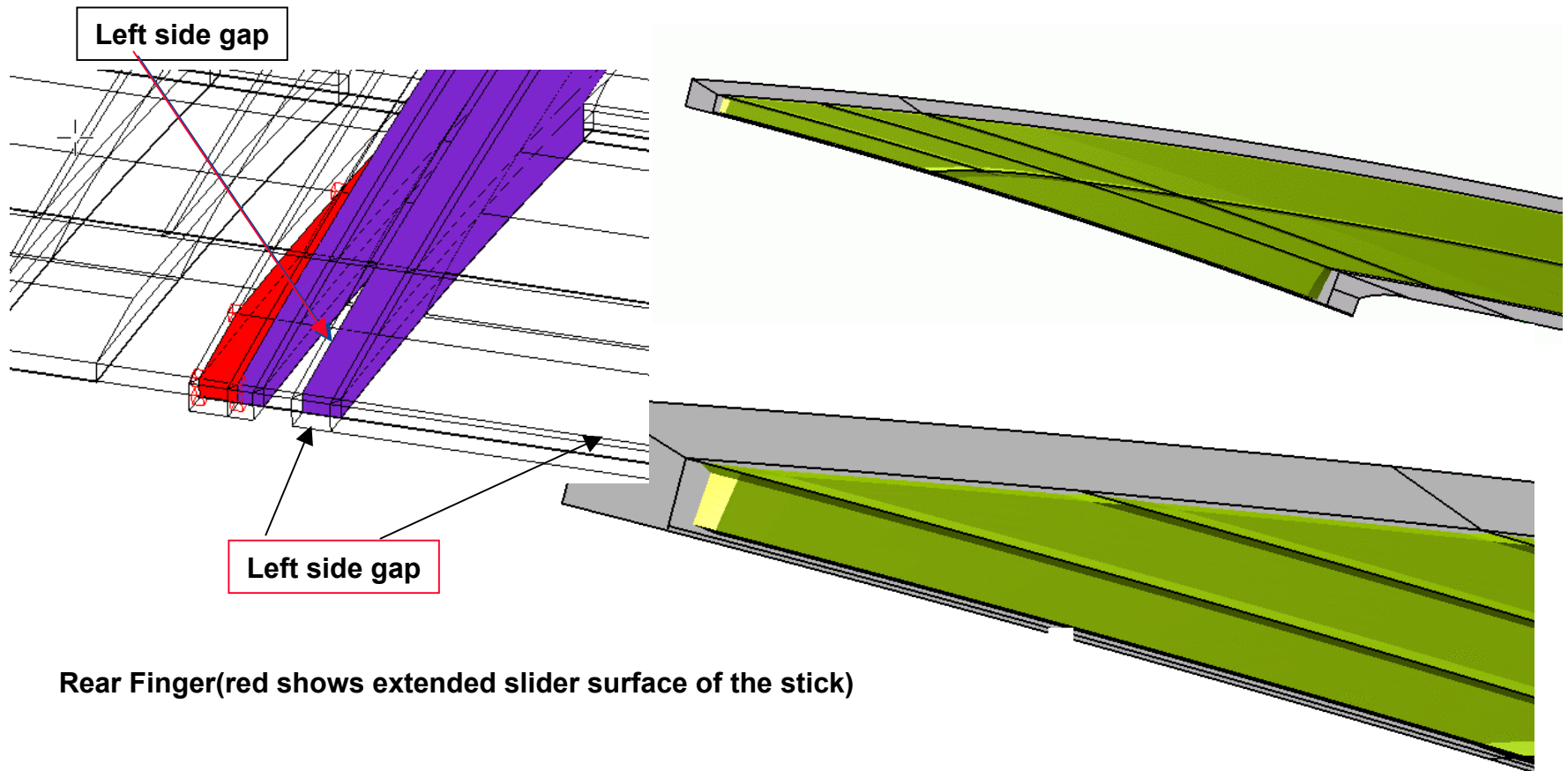
3d of computational domain





Domains identification and 3d view of the runner

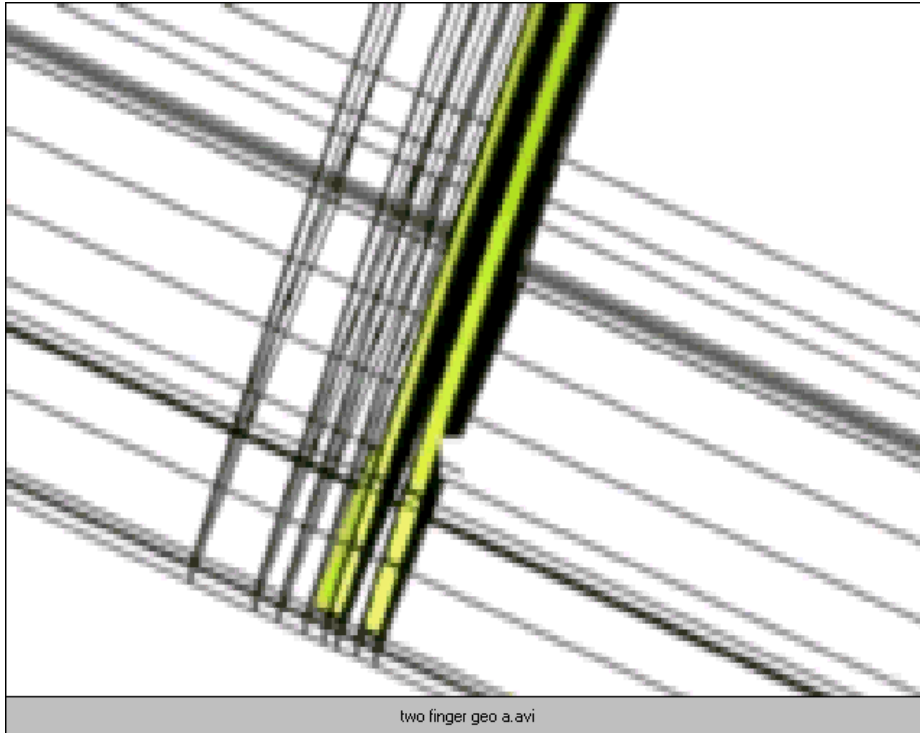




Rear Finger (red shows extended slider surface of the stick)



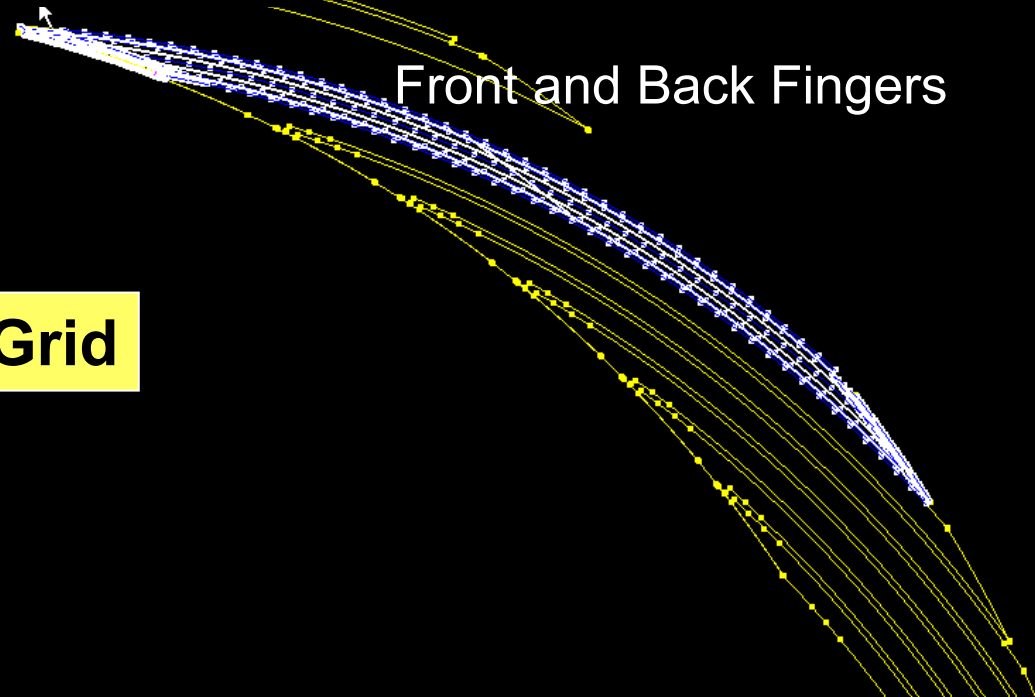
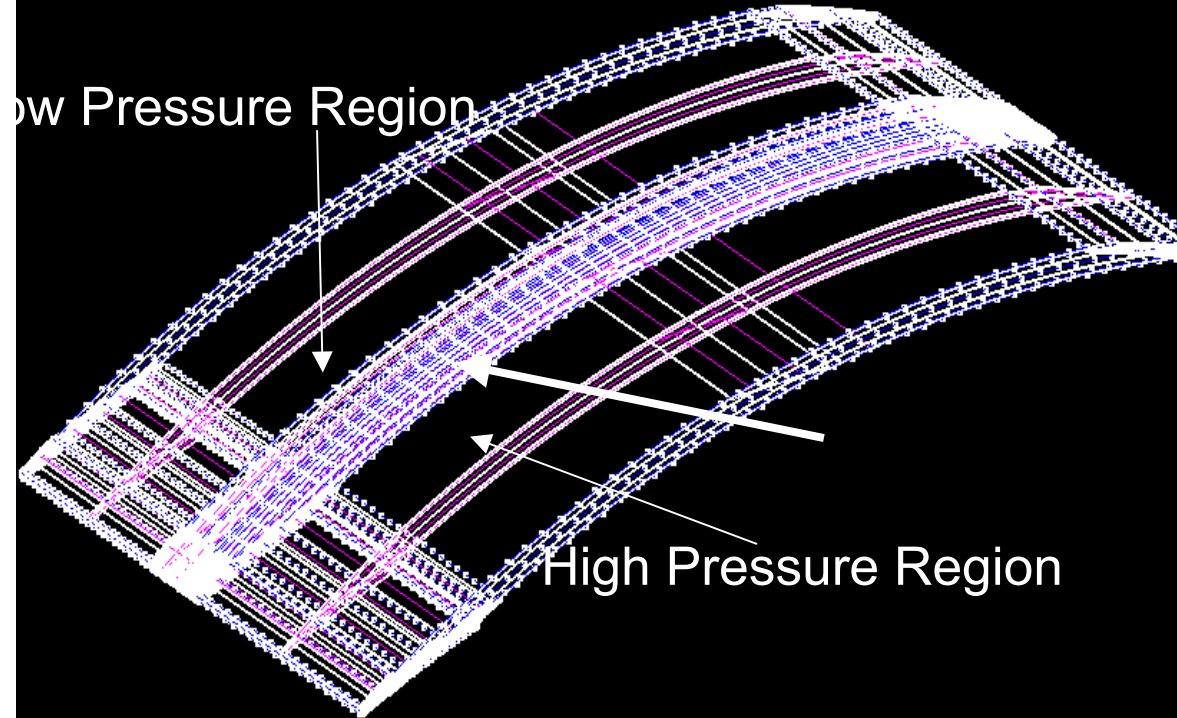
In line finger 3-D dimensional view the computational grid



two finger geo a.avi

This short clip shows the domains of the front high pressure finger and the low pressure finger behind it from different angles for a better geometrical understanding

Grid Details



Full Domain Computational Grid



- ⇒ **Mechanical model of the single finger and assembly of fingers.** FEMSTRESS is used for static and transient structural analysis, for both linear and nonlinear problems. The module is coupled with the other modules of CFD-ACE+ (flow, thermal, electrostatics, etc.) for multidisciplinary analyses. It uses a Fast and Efficient Vector Sparse Solver (VSS).
- ⇒ In the fluid solid interaction proposed by this study a two-way coupling will be utilized to proliferate shape changes due to deformation into computational flow domain. In this case for the flow within the finger seal domain, grid deformation and stress modules will be simultaneously activated, pressure forces integrated on all fingers, finger deflections will be calculated on each iteration and corresponding geometry/grid changes will be implemented.



⇒ Hydrodynamic fluid model. The computational engine used is CFD-ACE+, which is a product of CFD Research Corporation of Huntsville/Alabama. CFD-ACE+ supports structured, unstructured polyhedral, hybrid (structured/unstructured) moving grids and non-matching grid interfaces. It consists of pre-processor for grid generation (CFD-GEOM), GUI Module for model setup (CFD-GUI) and post-processor CFD-VIEW.

⇒ **Special features** that make CFD-ACE+ and FEMSTRESS especially suitable for our purposes are:

- coupled flow-structure interaction for steady-state and transient flow regimes
- solver stability which allow highly stretched nodes required for thin-film resolution
- ability to solve leakage flows in full Navier-Stokes formulation
- configurable GUI which allows creation of application specific templates/tools



RESULTS



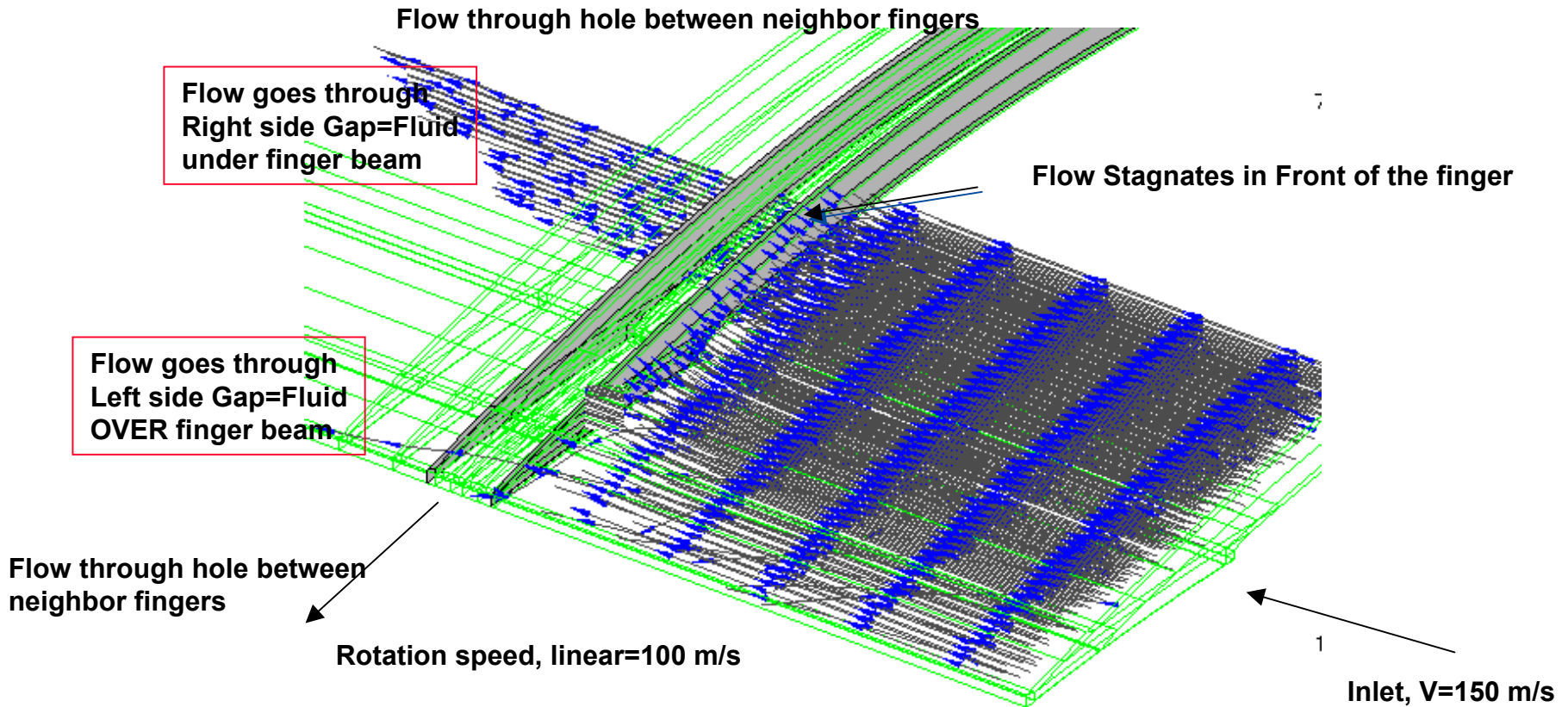
- ◆ **Flow in the Front Cavity**
- ◆ **Flow About the First Finger**
- ◆ **Flow in the Inter-Cavity First/Second Finger**
- ◆ **Flow Past The Second Finger (2nd Cavity) and Under the Pad**
- ◆ **Pad Motion**
- ◆ **Pressure Maps**



3D Example Solution (no deformation)



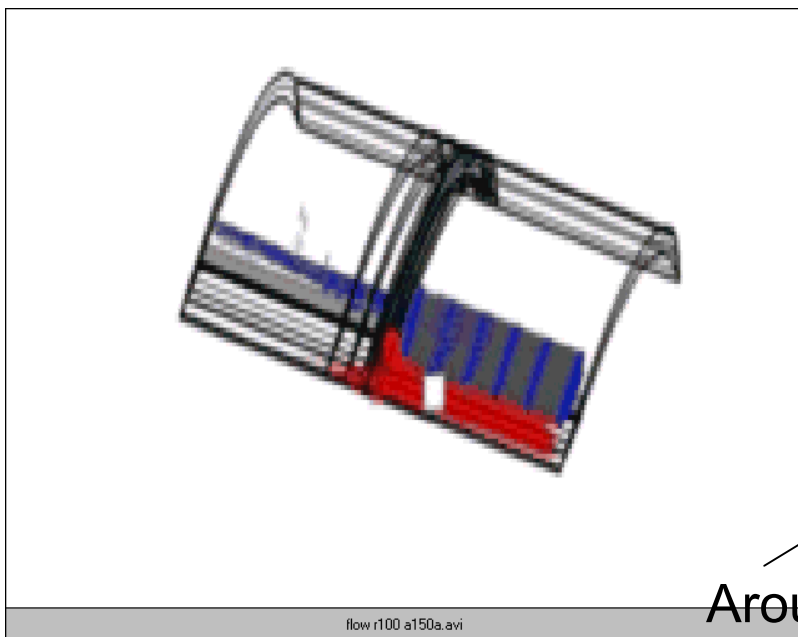
HIGH VELOCITY CASE –rotational flow is swung by axial





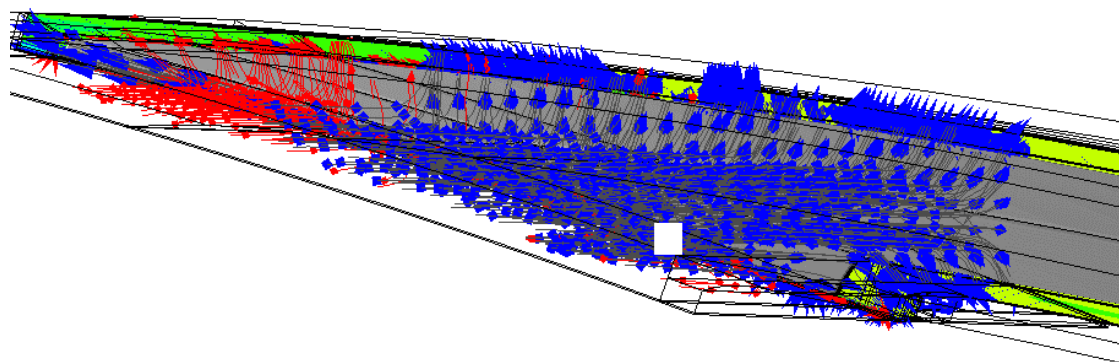
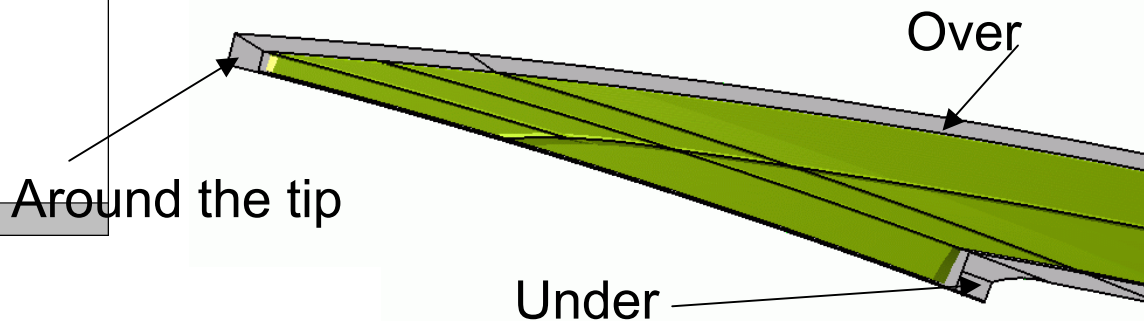
Flow in and around the Finger seal

$V_{\text{shaft}}=125 \text{ m/s}$, $V_{\text{inlet}}=100 \text{ ms}$ –no def



flow r100 a150a.avi

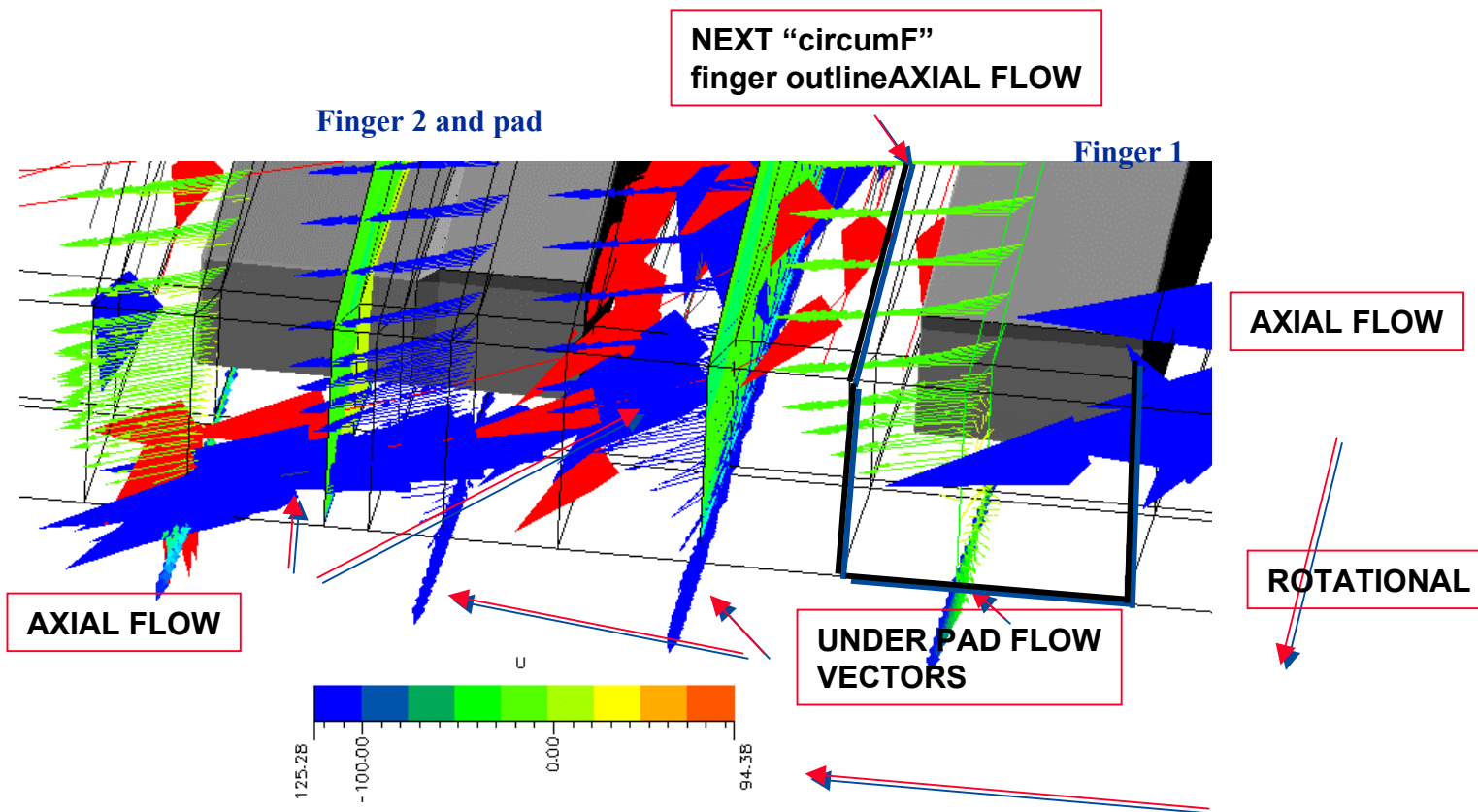
One can notice the blue arrows representing mostly flow over and under the finger, while the red arrows represent the flow around the tip of the finger through the gap left between two consecutive fingers.





Flow in and around the Finger seal-Details

$V_{\text{shaft}}=125 \text{ m/s}$, $V_{\text{inlet}}=100 \text{ ms}$ –no def

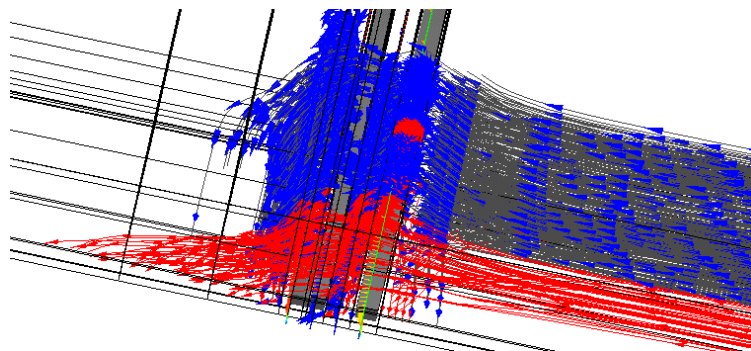


Notice that rotational flow in the gap (circumferential) is unaffected by flow through the seal.

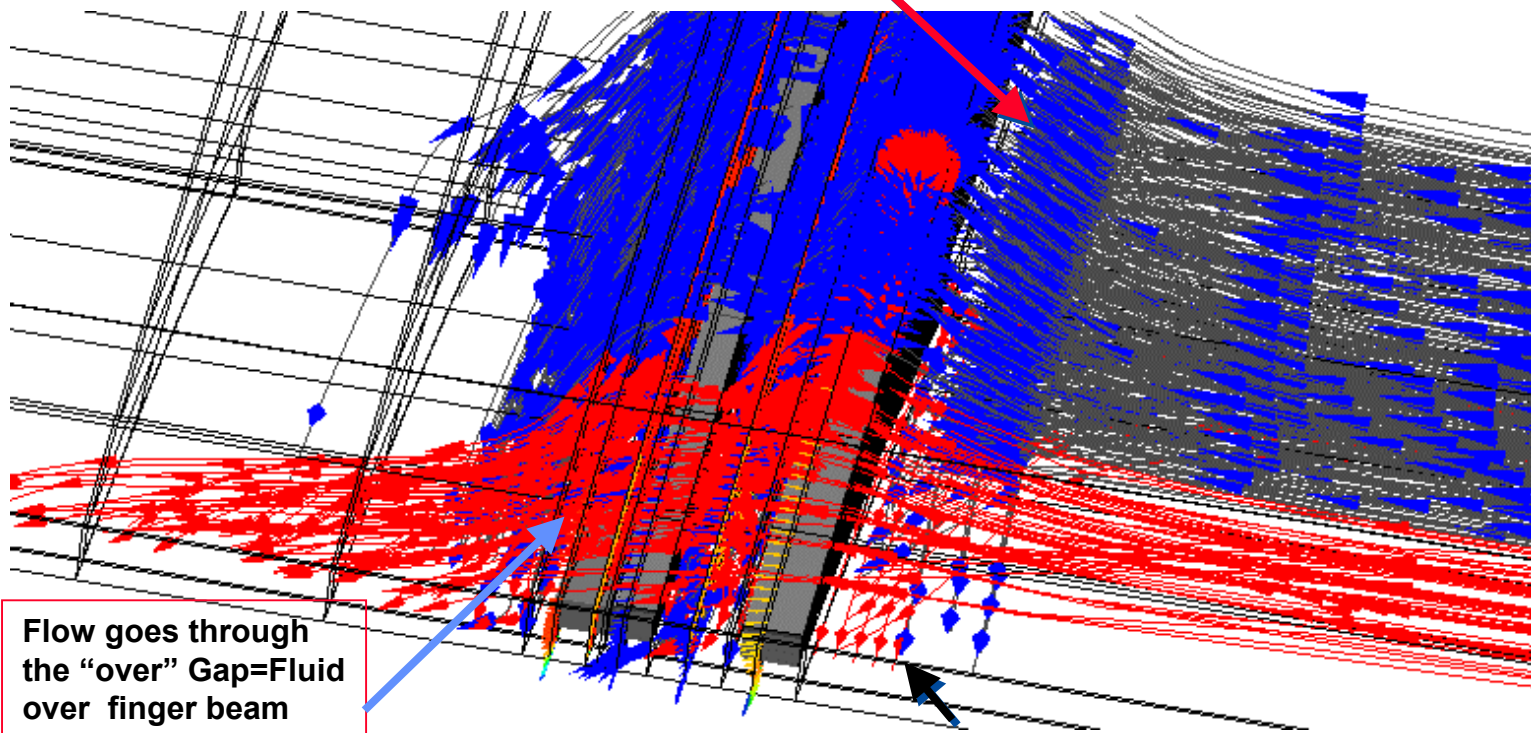
AXIAL FLOW—MAINLY IN THE "CIRCUMFERENTIAL GAP BETWEEN FINGERS---NOT UNDER THE PAD



3-D flows, $V_{\text{shaft}}=125 \text{ m/s}$, $V_{\text{inlet}}= 50 \text{ ms}$ no finger deformation



Flow goes through the “under” Gap
Fluid under finger beam



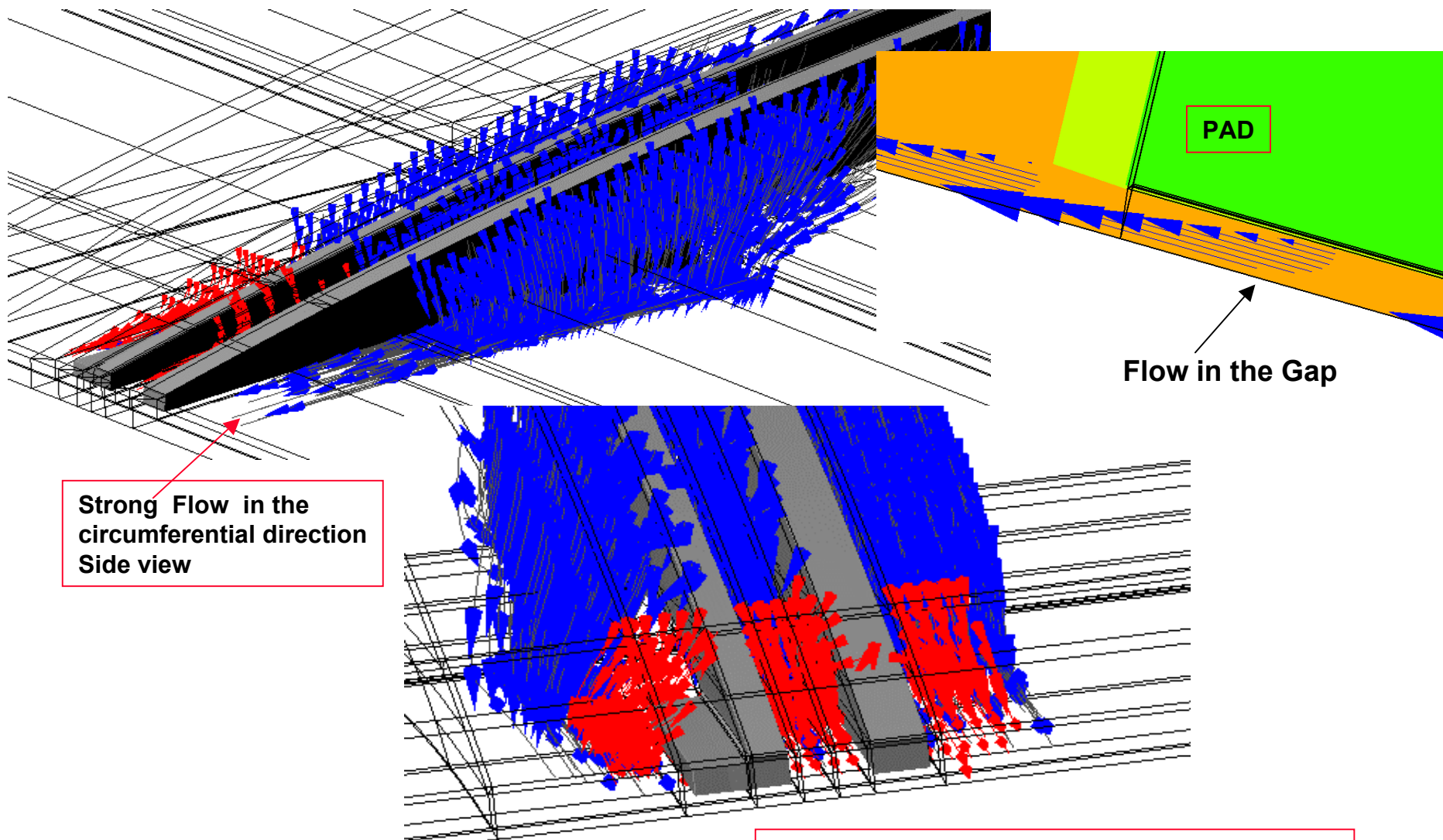
Flow goes through
the “over” Gap=Fluid
over finger beam

Circumferentially entrained flow



Rotational Dominated Flow

$V_{\text{shaft}}=100 \text{ m/s}$, $V_{\text{inlet}}=1 \text{ m/s}$, no def



Strong Flow in the circumferential direction
Side view

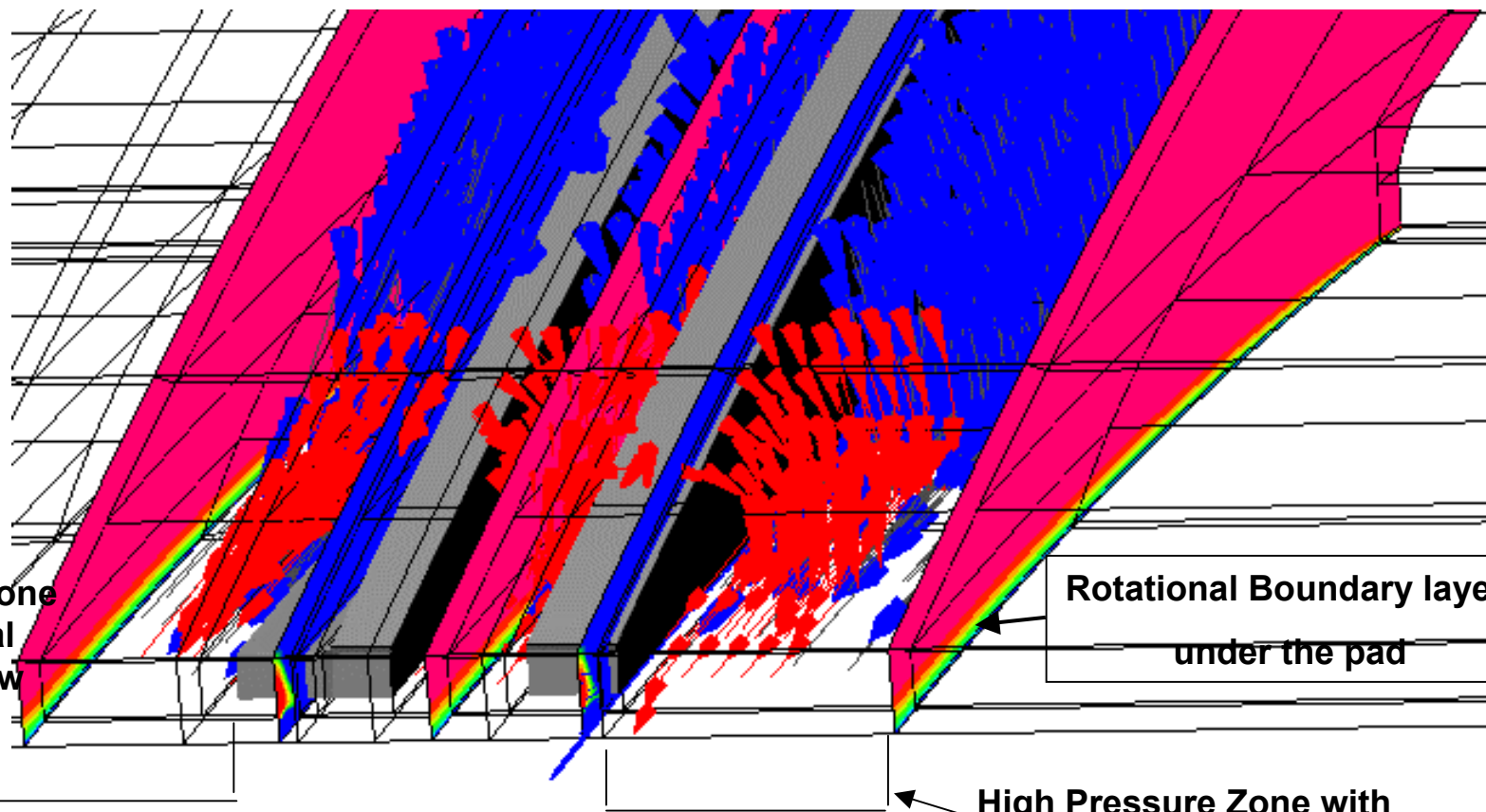
Flow in the Gap

Fingers Assembly and Flow Front View
Arrows pointing at us showing strong rotational (circumferential) flow



Rotational Dominated Flow-Cross Sections

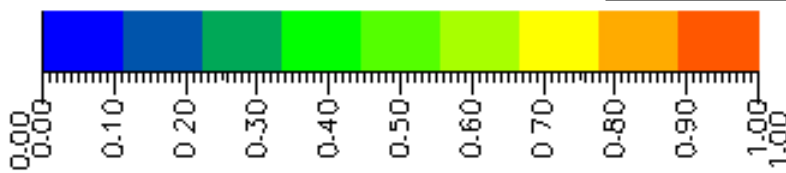
$V_{\text{shaft}}=100\text{m/s}$; $V_{\text{axial}}=1\text{m/s}$



Low Pressure Zone with rotational dominated flow

Rotational Boundary layer under the pad

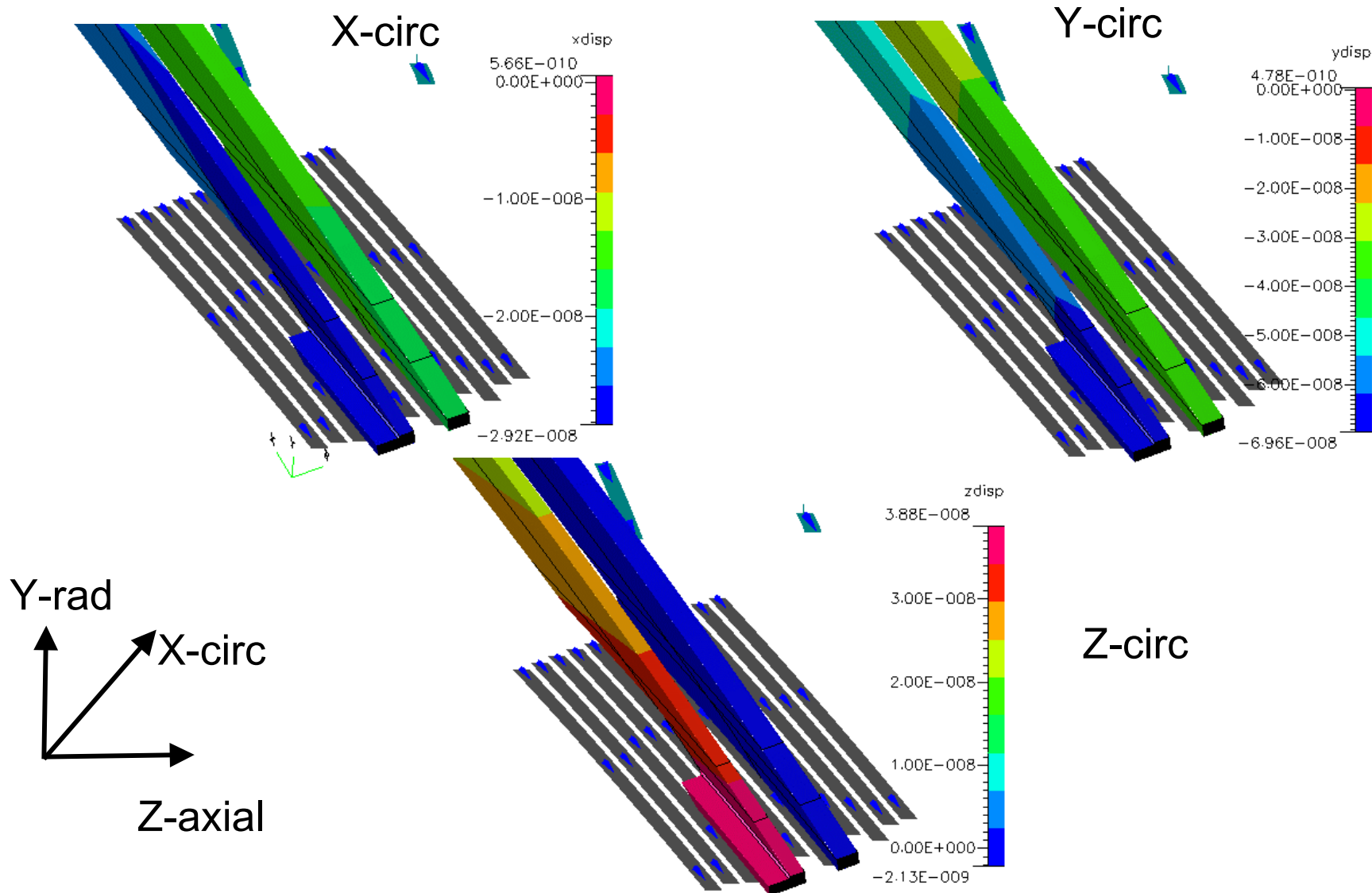
High Pressure Zone with rotational dominated flow





Color Coded displacements and Streamlines

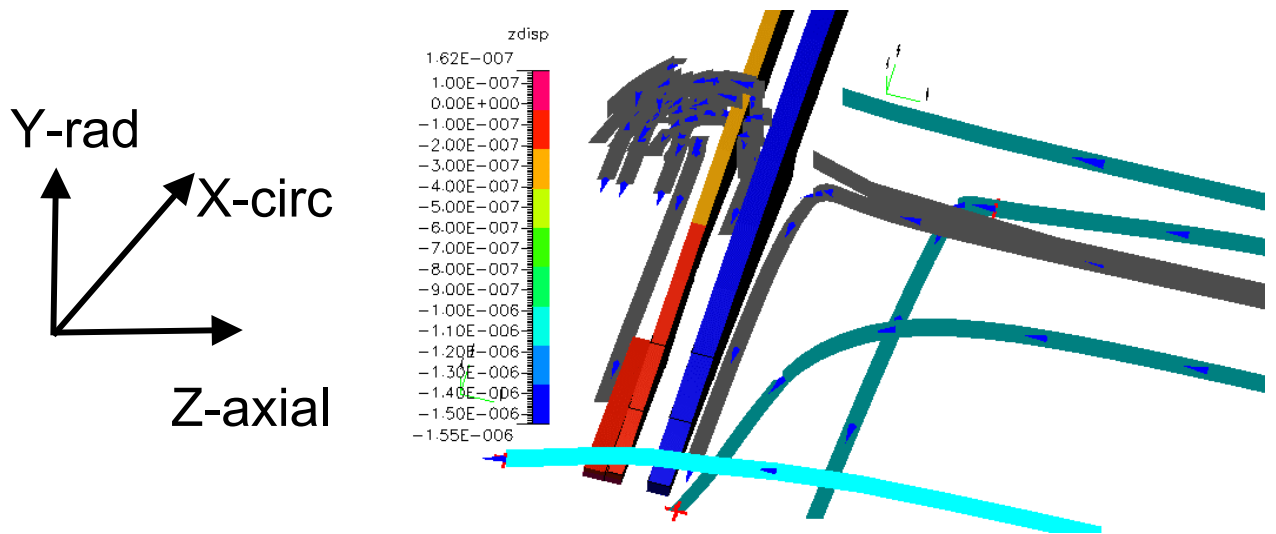
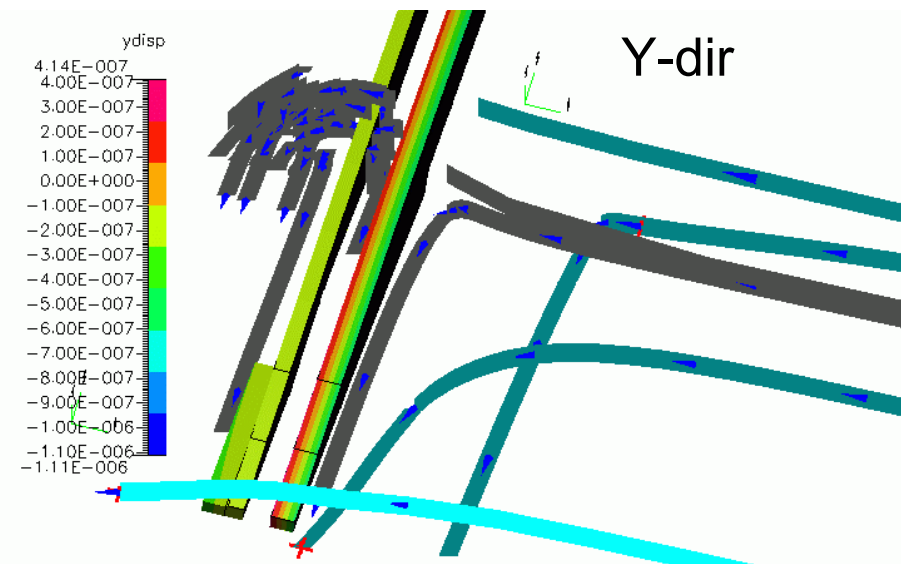
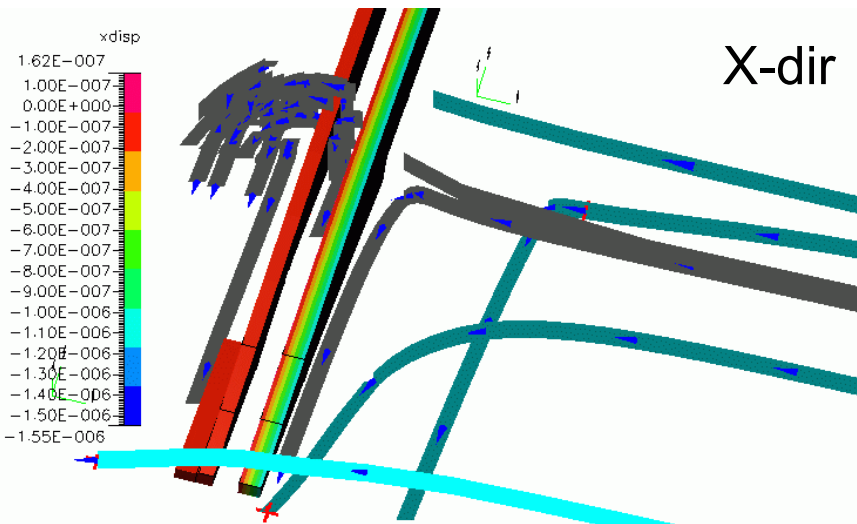
$V_{\text{shaft}}=100 \text{ m/s}$, $V_{\text{inlet}}=1 \text{ m/s}$

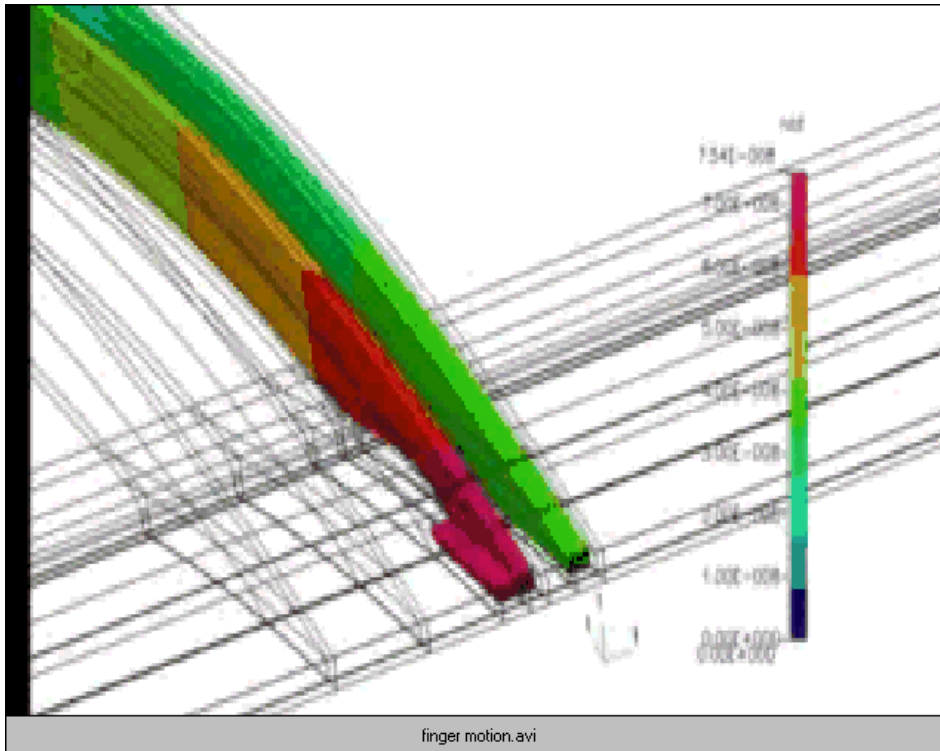




Color Coded displacements and Streamlines

$V_{\text{shaft}}=100 \text{ m/s}$, $V_{\text{inlet}}=50 \text{ ms}$

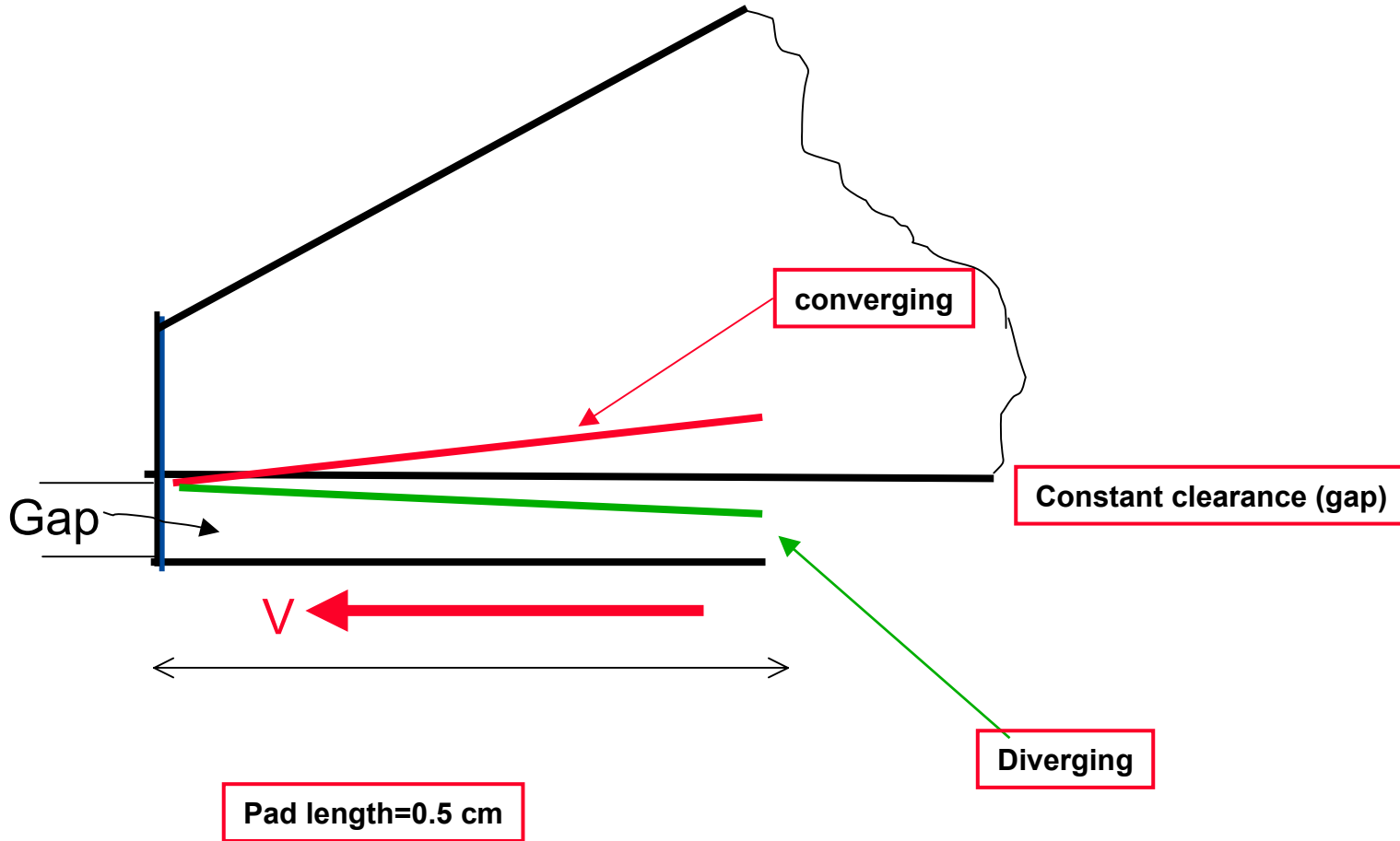




Brown-Red spectrum represents displacements in the positive direction while dark green to blue colors represent displacements in the negative direction

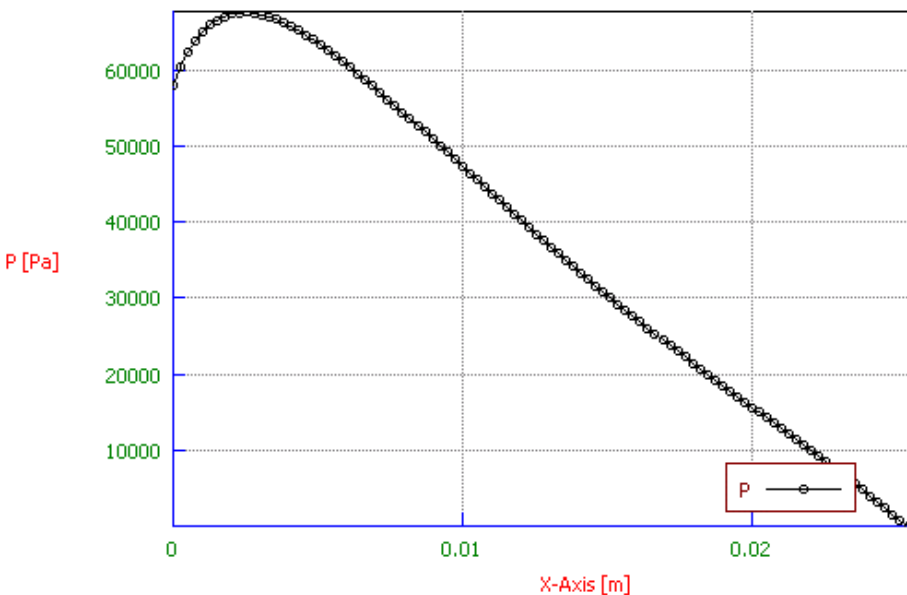


2-D Pressure Distributions for Characteristic Gap

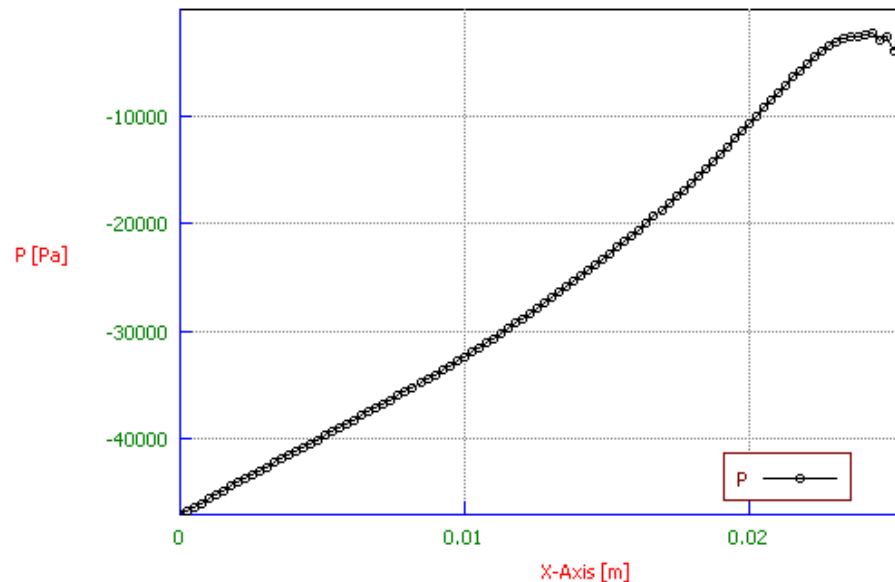




PRESSURE DISTRIBUTION: 1 inch long pad, 1 mil to 0.5 mil converging gap



PRESSURE DISTRIBUTION: 1 inch long pad, 0.5 mil to 1 mil diverging gap





Conclusions



- ◆ **Good insights into flow formation, finger motion, pressure development.**
- ◆ **Proper software well chosen. FEMSTRESS + CFD ACE+**
- ◆ **A LOT MORE WORK TO DO.**